

Review on Performance of Dual Mass Flywheel over Conventional Flywheel

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Abstract

The Dual Mass Flywheel is used for dampening the oscillations created at the time of working in automotive power transmission system. Dual mass flywheel is a multi-clutch device. The torsional frequency is defined as the rate at which the torsional vibration produced. When the torsional frequency of the crankshaft is equal to the transaxles torsional frequency an effect known as the torsional resonance occurs. When the operating speed of the engine is minimum, vibration occurs due to the torsional resonance and this can be avoided using dual mass flywheel. We explained detailed initial model in the paper. In this mainly having the two arc springs and two masses in the DMF and their nature. By experiment the Dual Mass Flywheel model is compared to convention flywheel. Finally the observation of the engine torque using the DMF is taken. The DMF is manufactured and done experiment or testing to see the results. And then results which came are compare with the conventional flywheel.

Keywords— Dual & conventional mass flywheel, concept, study.

I. INTRODUCTION

Use of vehicle is increases in now days and the field development is growing rapidly so higher-performance engines paralleled by an increased demand for driver comfort. In addition, lean concepts, extremely low-speed engines and new generation gear-boxes using light oils contribute to this. Since the year 1980s, this advancement has pushed the classic torsion damper as an midway integral part of the clutch driven plate to its limits. With the same or even less installation space available, the classic torsion damper has proved constantly increasing engine torques. Extensive development in this field resulted in a simple, but very effective solution –DMF a new torsion damper concept for the drive train. All the referred papers refer to various regeneration methods of flywheel by which we can produce power and store in battery for further use, or method and implements to reduce weight of the flywheel using different composite materials. In our case using the two spring two mass system to produce useful vibrations which will be employed to increase the inertia of the system and thereby enable us to either reduce the weight of existing flywheel or increase power output using existing weight of flywheel also we will be able to improve acceleration characteristic of given system. In today's world power train control systems need accurate torque information to perform various tasks. These tasks include for example the clutch actuation in automated manual transmissions (AMTs) and dual-clutch transmissions (DCTs) as well as the control of electric motors in hybrid power trains. Indirect torque estimation is needed because the direct measurement of the transmitted torque using strain gages cannot be done in volume

production cars for economic reasons. One source for power train torque estimation is the engine itself. However, the torque estimation provided by the internal combustion engine is based on complex thermodynamic models. The repeated period combustion cycles of a 4-stroke engine produce torque fluctuations which excite torsional vibration to be passed down the drive train. The resulting noise and vibration, such as gear rattle, body boom and load change vibration, result in poor noise behavior and driving comfort. Thus engine models tend not to be reliable in all situations. Because of rotational vibrations created due to variation in torque this variation results from the discrete piston combustion cycle of the engine as a function of the ignition frequency. Here the possibility of torque estimation using the Dual Mass Flywheel is analyzed. Such that DMF is used to detect engine misfire which is similar to power train torque observation. The objective when developing the Dual Mass Flywheel was therefore to isolate torsional vibration from the drive train as much as possible caused by the engine's rotating mass. The Dual Mass Flywheel almost absorbs this torsional vibration. The result is very good vibration damping.

II. LITERATURE REVIEW

1. Jordan Firth, Jonathan Black

This paper explains the vibration interaction in a multiple flywheel system. Flywheels can be used for kinetic energy storage. In this paper one unstudied problem with vibration interaction between multiple unbalanced wheel. This paper uses a linear state space dynamics model to study the impact of vibration interaction. Specifically, imbalanced induced vibration inputs in one flywheel rotor are used to cause a resonant whirling vibration in another rotor. Vibration is most severe when both rotors are spinning in the same direction.^[4]

2. Bjorn Bolund, Hans Bernhoff, Mats Leijon

This paper explains the use of flywheel. Nowadays flywheels are complex construction where energy is stored mechanically and transferred to and from the flywheel by an integrated motor or generator. The wheel has been replaced by a steel or composite rotor and magnetic bearings have been introduced. By increasing the voltage, current losses are decreased and otherwise necessary transformer steps become redundant.^[5]

3. Ulf Schaper, Oliver Sawodny, Tobias Mahl and Uti Blessing

They explain the DMF along with its application and components. Afterwards a detailed model of the DMF dynamics is presented. This mainly includes a model for the two arc springs in the DMF and their friction behaviour. Both centrifugal effects and redirection forces act radially on the arc spring which induces friction. The numerical method is used to measure model validation.^[6]

III. PROBLEM STATEMENT

To inspect the performance characteristics of Dual mass flywheel over conventional flywheel. And compare which one having good efficiency for use. Optimize the Dual mass flywheel performance results than that of conventional flywheel.

IV. DESIGN OF CONVENTIONAL FLYWHEEL

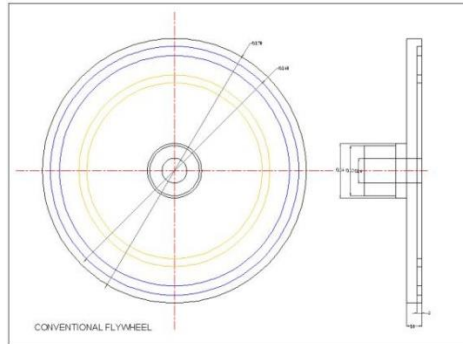


Figure no.1 Conventional Flywheel

V. DESIGN OF DUAL MASS FLYWHEEL SYSTEM

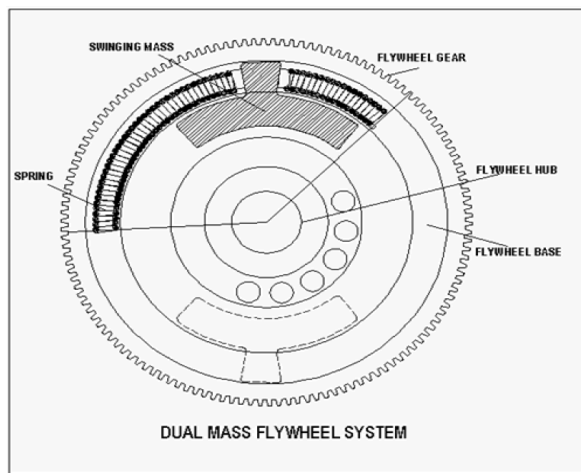


Fig.5.Dual Mass Flywheel System

Figure no.2 Dual Mass Flywheel

VI. EXPERIMENTAL SETUP[ref.1]



Figure no.3 Experimental Setup

VII. EXPERIMENTAL RESULT

The effect of inertia can be seen by the difference in the fluctuation of energy in the DMF and the Conventional flywheel.[1]

Let, Maximum fluctuation of energy of Dual mass flywheel = $\nabla E_{dmf} = mR^2 \omega_{dmf}^2 Cs$

Where,

$$R = \text{Mean Radius of rim} = 68 \text{ mm} = 0.068$$

ω_{dmf} = mean angular speed of dual mass flywheel

$$= \frac{2\pi(N_1 + N_2)}{2} = \frac{2\pi(1430 + 930)}{2}$$

$$\omega_{dmf} = 7414 \text{ rad/sec}$$

C_s = Coefficient of fluctuation of speed = $\frac{N_1 - N_2}{N}$

$$\text{Where } N = \frac{(N_1 + N_2)}{2} = 1180$$

$$C_s = 1430 - 930 / 1180 = 0.423$$

$$\nabla E_{dmf} = mR^2 \omega_{dmf}^2 Cs = 1.9 \times 0.068^2 \times 7414^2 \times 0.423 = 204.27 \text{ KJ}$$

Maximum fluctuation of energy of Conventional flywheel

Where, m = mass of flywheel = 1.9Kg

$$R = \text{Mean Radius of rim} = 68 \text{ mm} = 0.068$$

ω_{cnv} = mean angular speed of dual mass flywheel

$$= \frac{2\pi(N_1 + N_2)}{2} = \frac{2\pi(1315 + 910)}{2}$$

$$\omega_{cnv} = 6990 \text{ rad/sec}$$

C_s = Coefficient of fluctuation of speed = $\frac{N_1 - N_2}{N}$

$$\text{Where } N = \frac{(N_1 + N_2)}{2} = 1112$$

$$C_s = 1315 - 910 / 1112 = 0.364$$

$$\nabla E_{cnv} = mR^2 \omega_{cnv}^2 Cs = 1.9 \times 0.068^2 \times 6990^2 \times 0.364 = 156.25 \text{ KJ}$$

$$\text{Effectiveness } (\epsilon) = \frac{\nabla E_{dmf}}{\nabla E_{cnv}} = 204.27 / 156.25 = 1.30$$

Thus the Dual mass flywheel is 1.3 times effective than the Conventional flywheel

VIII. OBSERVATIONS

Sample calculations:

a)

$$\begin{aligned} \text{output torque} &= \\ W \times 9.81 \text{ radius of dyno brake pulley} \\ T_{op} &= 4 \times 9.81 \times 0.032 = 1.26 \text{ Nm} \end{aligned}$$

b)

$$\begin{aligned} \text{Efficiency} &= \\ (\text{Output power} / \text{input power}) \times 100 &= \\ 152.39 / 205 &= 74.33 \end{aligned}$$

RESULT TABLE

Table no. 1 Result Table of Conventional Flywheel[ref.1]

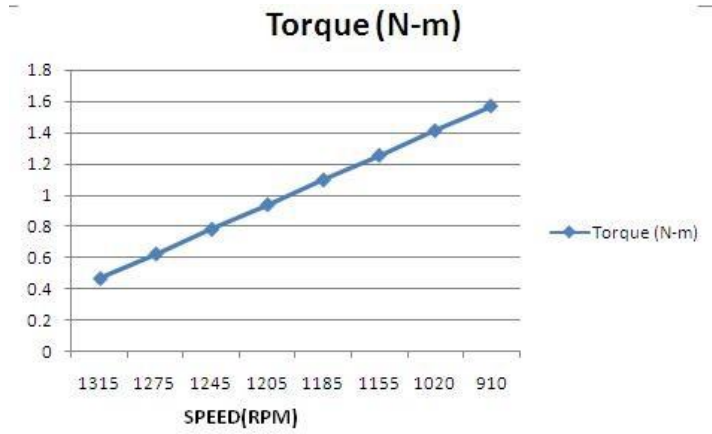
Sr. No	Load(gm)	Speed(rpm)	Torque(Nm)	Efficiency
1	1500	1315	0.47088	31.6349
2	2000	1275	0.62784	40.8969
3	2500	1245	0.7848	49.9183
4	3000	1205	0.94176	57.9773
5	3500	1185	1.09872	66.5176
6	4000	1155	1.25568	74.0955
7	4500	1020	1.41264	73.6144

Table No.2 Result Table of Dual Mass Flywheel[ref.1]

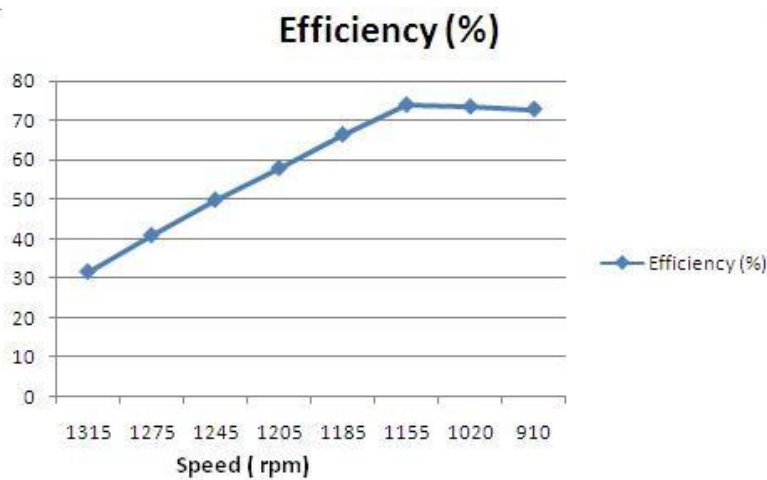
Sr. No	Load(gm)	Speed(rpm)	Torque(Nm)	Efficiency
1	1500	1425	0.47088	34.2594
2	2000	1345	0.62784	43.1147
3	2500	1365	0.7848	54.6948
4	3000	1315	0.9417	63.2256
5	3500	1285	1.09872	72.085
6	4000	1245	1.25568	79.8184

7	4500	1080	1.41264	77.895
8	5000	930	1.5696	74.5292

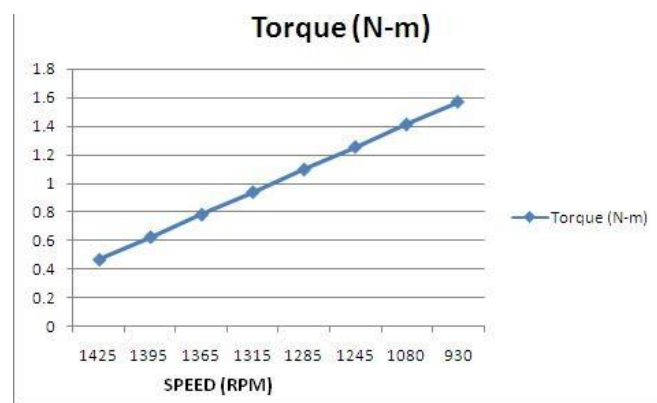
Graph of Torque Vs Speed for Dual Mass Flywheel



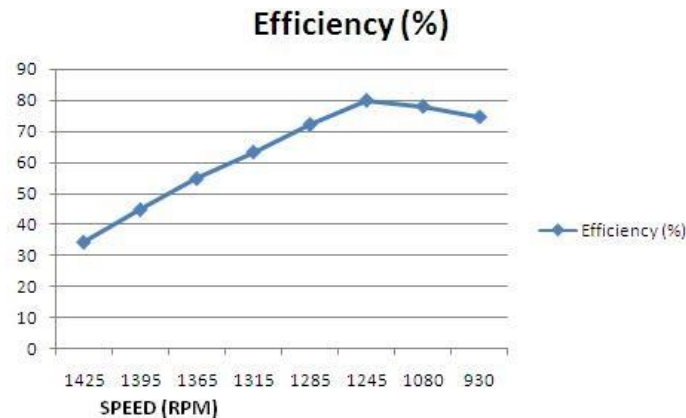
Graph of Efficiency Vs Speed for Dual Mass Flywheel



Graph of Torque Vs Speed for Conventional Flywheel

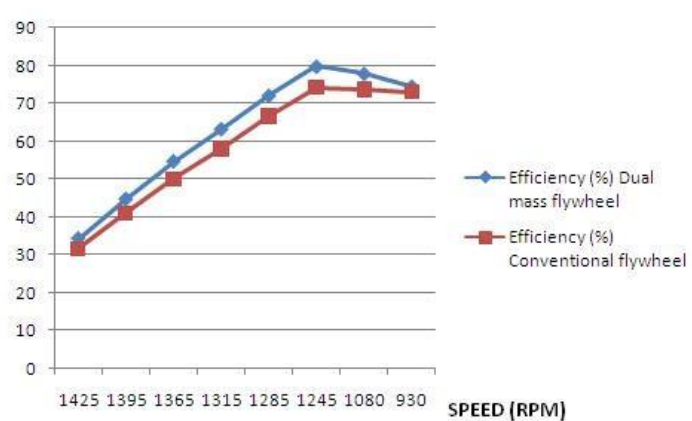


Graph of Efficiency Vs Speed for Convention Flywheel



IX. CONCLUSION

Comparison- Efficiency of Conventional and Dual mass flywheel:



The DMF having good efficiency than that of Conventional flywheel, it is near to 5 to 6% efficient. Due to this increases fuel economy of engine.

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