

## Analysis of Helical Coil Heat Exchanger Using CFD

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### Abstract

Helical coil has better heat transfer rate as compared to shell and tube heat exchanger, Because of development of secondary flow. There are so many applications where the helical coil is used such as heating ventilation and air conditioning applications, steam generators used in steam power plants, in condenser's, the reason behind using helical tube is it provides large surface area per unit volume. In the presented work the heat transfer coefficient ( $h_i$ ) for inside and heat transfer coefficient ( $h_o$ ) for outside from the different research paper were studied and compared. For the calculation of heat transfer coefficient MATLAB code is developed for the same. The values of heat transfer coefficient for inner side has agreement between each other, however outside heat transfer coefficient has no agreement is found. Computational fluid dynamics is used for the simulation and results were compared with experimental and they found in close agreement.

**Keywords-** Helical Coil, CFD, Matlab, Heat Transfer Coefficient

## NOMENCLATURE

$d_i$	Tube inner diameter ( $=2*r$ ) in m
$d_o$	Tube outer diameter in m
$D_{c,b}$	Coil bottom diameter ( $=2*R_{c,b}$ ) in m
$D_{c,t}$	Coil top diameter ( $=2*R_{c,t}$ ) in m
$D$	Diameter of straight helical coil ( $=2*R$ ) in m
$D_{ave}$	Average diameter ( $=D_{c,b} + D_{c,t}/2$ ) in m
$D_{s,o}$	Outer shell diameter in m
$D_{s,i}$	Inner shell diameter in m
$L$	Length of coil in m
$H$	Height in m
$P$	Pitch of coil in m
$N$	Number of turns
$B$	Clearance in m
$k_t$	Thermal conductivity of tube material in $W/m \text{ } ^\circ K$
$k_c$	Thermal conductivity of tube fluid in $W/m \text{ } ^\circ K$
$k_s$	Thermal conductivity of shell fluid in

	W/m °K
$\beta_c$	Coefficient of volumetric thermal expansion of tube fluid in 1/°K
$\beta_s$	Coefficient of volumetric thermal expansion of shell fluid in 1/°K
$\rho_c$	Mass density of tube fluid in (kg/m <sup>3</sup> )
$\rho_s$	Mass density of shell fluid in (kg/m <sup>3</sup> )
$\mu_c$	Dynamic viscosity of tube fluid in m/kgS
$\mu_s$	Dynamic viscosity of shell fluid in m/kgS
$T_{c,i}$	Inlet temperature of tube fluid in °K
$T_{c,o}$	Outlet temperature of tube fluid in °K
$T_{s,i}$	Inlet temperature of shell fluid in °K
$T_{s,o}$	Outlet temperature of tube fluid in °K
$C_{p,c}$	Specific heat of tube fluid in J/Kg°K
$C_{p,s}$	Specific heat of shell fluid in J/Kg°K
$C_{p,mn}$	Minimum specific heat in J/Kg°K
PF	Parallel flow arrangement
CF	Counter flow arrangement
$m_c$	Mass flow rate of tube fluid in Kg/ S
$m_s$	Mass flow rate of shell fluid in Kg/ S
$(mC_p)_{min}$	Minimum value of product of m and $C_p$
De	Dean Number
$D_{eq}$	Equivalent Diameter
$D_{hx}$	Hydraulic Diameter
DR	Diameter Ratio
PR	Pitch Ratio
HCCH E	Helical cone coil heat exchanger
HCCS S	Helical cone coil straight shell
HCCC S	Helical cone coil cone shell
$\theta$	Cone angle of conical coil
N	Velocity exponential
$R_{th}$	Thermal Resistance
Z	drag coefficients
	<b>SUBSCRIPTS</b>
C	Cold water
H	Hot water

I	Inner, tube side
S	Shell
T	Tube
C	Coil
O	Outer, outside
Ov	Overall
Ave	Average
W	Wall

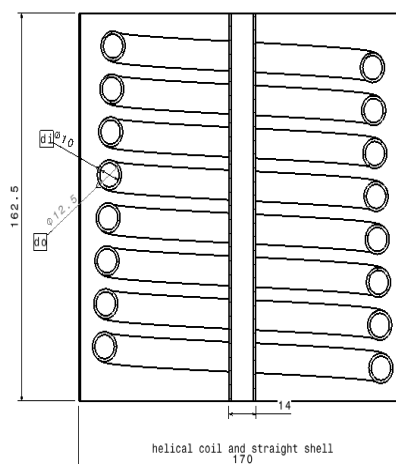
## 1.0 INTRODUCTION

### Straight helical coil heat exchanger

“Heat exchanger is a device which is use to transfer the heat from one fluid to another fluid through the same device”. In helical coil heat exchanger the helical coils are used for the heat transfer. There are so many applications where the helical coil is used such as heating ventilation and air conditioning applications, steam generators used in steam power plants, in condenser’s, the reason behind using helical tube is it provides large surface area per unit volume. Although such heat exchanger provides high heat transfer rates, we are finding very less research in this area. We are not able to find information about heat transfer coefficient for natural convection for helical coils. But we all are well known about the heat transfer coefficient are available for natural convection for vertical plate and horizontal plate. This lacuna provide motivation for doing research in this area to fill the present need.<sup>[1]</sup>

From the research paper it is confirmed that heat transfer from the helical coil is high as compared to straight tube heat exchanger because of the generation of secondary flow of fluid in plane normal to the main flow in the helical tubes which show great performance in heat transfer enhancement while the uniform curvature of spiral conical structure inconvenient in pipe installation in heat exchanger.<sup>[2]</sup>

In this type of heat exchanger, because of the centrifugal action the secondary flow is generated which acts in plane normal to the main flow in the helical tubes. as the maximum velocity is at the centre, hence the centrifugal force is maximum at centre and the fluid at the centre is subjected to this maximum centrifugal action, and that’s why fluid pushes towards the outer wall. The fluid present near the outer wall replaces the fluid ejected outwards while moving in-words along the tube wall, which in-turns forms the two symmetrical vortices about a horizontal plane through the tube center.



**Fig 1.1 Straight Helical Coil Heat Exchanger**

Inside heat transfer coefficient for helical coil and curved tube are greater than inside heat transfer coefficient of straight tube because of secondary flow (Dean Vortex) in curved tube and it is characterized by Dean Number which is equal to

$$De = Re \times ((d_i/D)^{0.5}) \quad (1.1)$$

In this type the curvature ratio is constant. Secondary flow become intensive, which in turn increases (hi).

For calculation of outside heat transfer coefficient ( $h_0$ ) correlations are found only for typical applications and specified ranges of Re, Ra study by researchers.

Generally correlations for  $h_0$ , covering entire range of Re,  $d_i/D$  is not found due to this we have used the available correlations of straight tube.

It is observed that the flow is in viscous regime for inside coiled tube up to much higher Reynolds number than that for straight tubes. Helical coils are having high heat and mass transfer when it compared to straight tube because of the generation of secondary flow in the primary flow. [3]

## 2.0 LITERATURE SURVEY

The Mohamed Ali<sup>[1]</sup> was performed the experimental investigation of Natural convection made to study, steady type Natural Convection was obtained from turbulent natural convection to water. The experiment have been carried for four coil diameter to tube diameter ratio for five and ten coil tubes and for five pitch outer diameter ratio.

He correlated Rayleigh Number for two different coil sets and the heat transfer coefficient decreases with coil length for tube diameter  $d_o = 0.012\text{m}$  but increases with coil length for  $d_o = 0.008\text{m}$ . For tube diameter of 0.012 m with either five or ten coil turns, critical  $D/d_o$  is obtained for a maximum heat transfer coefficient.

Yan ke<sup>[2]</sup> investigated numerical simulation of conical tube bundles. He observed the effect of structural parameters on heat transfer characteristics. fluid flow characteristics inside tube of different cross section also investigated result shows that cone angle cross section have been significant effect inside heat transfer. Also helical pitch has little influence on heat transfer enhancement. He also includes that the secondary fluid become intensive along the tube due to increase of tube curvature. Secondary fluid flow contains four contours and flow direction of each contour are different due to this heat transfer rate increases.

J.S Jaykumar<sup>[3]</sup> after validating the methodology of CFD analysis of a heat exchanger, the effect of considering the actual fluid properties instead of a constant value is established. Various boundary conditions are compared to calculate heat transfer characteristics inside a helical coil. It is found that the specification of a constant temperature or constant heat flux boundary condition for an actual heat exchanger does not give satisfactory result through modelling. For this problem the heat exchanger is analysed with considering conjugate heat transfer and properties of heat transport fluid which are temperature dependent. An experimentation was carried out for the calculation of the heat transfer characteristics. Experimental results and CFD calculation results using the CFD package FLUENT 6.2 are compared. Finally the correlation is developed by using the experimental result obtained. The inner heat transfer coefficient of the helical coil is thus obtained. CFD code FLUENT is used for finding Heat transfer characteristics of the heat exchanger with helical coil. The CFD predictions are in good agreement with the experimental results within experimental error limits.

N. Ghorbani<sup>[4]</sup> conducted experimental study of thermal performance shell and coil heat exchanger in the purpose of this article is to access the influence of tube diameter, coil pitch, shell side and tube side mass flow rate on the modified effectiveness and performance coefficient of vertical

helical coiled tube heat exchanger. The calculation has been performed for the steady state and the experiment was conducted for both laminar and turbulent flow inside coil. It was found that the mass flow rate of tube side to shell ratio was effective on the axial temperature profiles of heat exchanger. He concluded that with increasing mass flow rate ratio the logarithmic mean temperature difference was decreased and the modified effective's decreases with increasing mass flow rate.

R. Patil [5] suggested design methodology for helical coil heat exchanger. heat transfer coefficient based on the inside coil diameter  $h_i$ , is obtained using method for a straight tube either one of Sieder –Tate relationships or plot of the Colburn factor  $J_H$  vs  $Re$ . outside heat transfer coefficient is calculated using correlation for different range of Reynolds number. Helical coil heat exchanger is the better choice where space is limited and under the conditions of low flow rates or laminar flow.

### 3.0 SIMULATION

#### 3.1 Geometry and Meshing

Geometry is created in the CATIA V5 for both the parallel and counter flow. After that the meshing of geometry is completed. Both volume and surface mesh is done for the geometry.

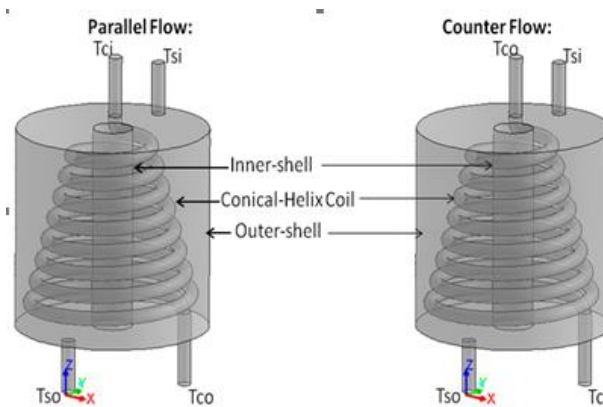


Fig.3.1 Geometry

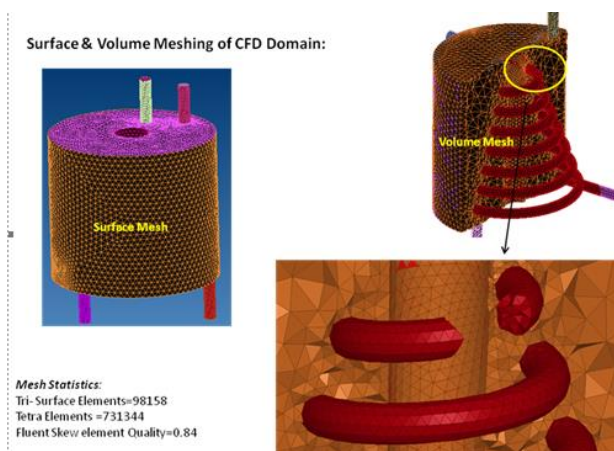


Fig.3.2 Meshing

### 3.2 Boundary Conditions & Assumptions:

- **Solver :** Pressure based Steady State
- **Model:** K-epsilon Realizable Standard Wall Function
- **Boundary Conditions:**
  - Fluid: water-liquid
  - Coil inlet=mass flow inlet type with Temp Tci
  - Shell inlet=mass flow inlet type with Tsi
  - Coil outlet= static pressure outlet
  - Shell outlet= static pressure outlet
  - Wall-coil= copper, 0.12 mm thick, coupled wall
  - All other walls= stationary no slip

## 4.0 RESULTS

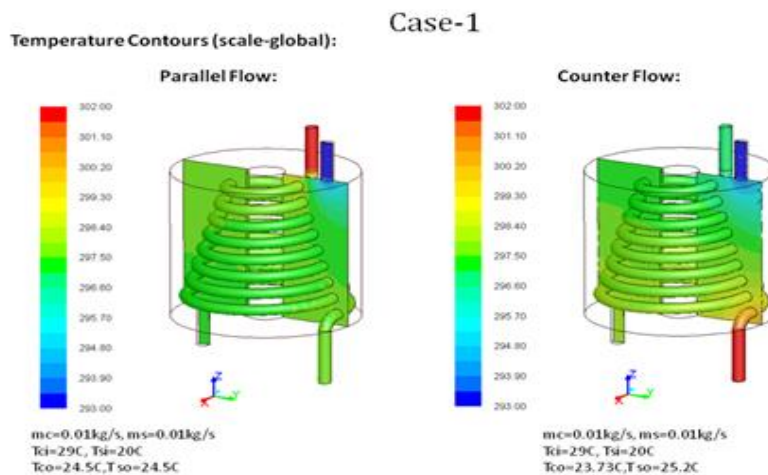
### 4.1 Temperature Results Of CFD Compared With Experimental

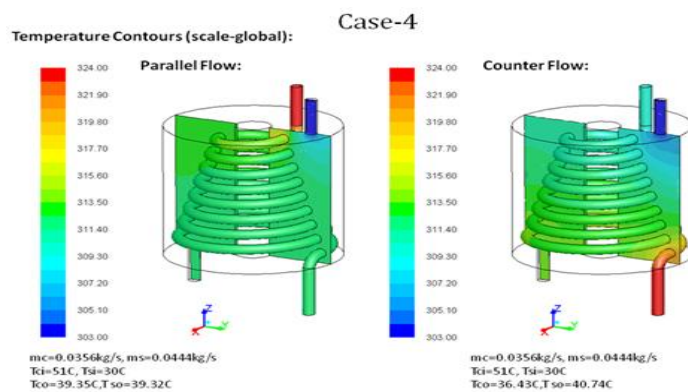
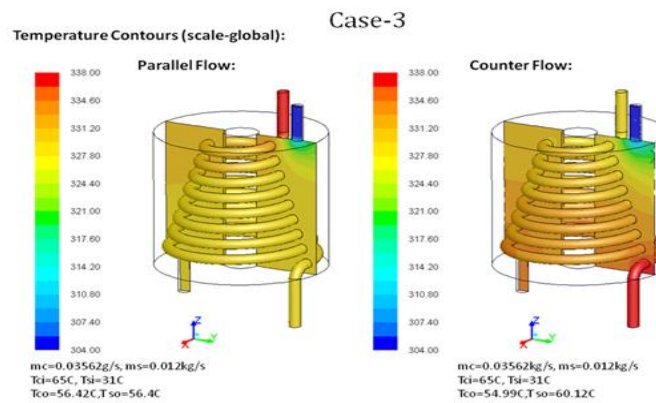
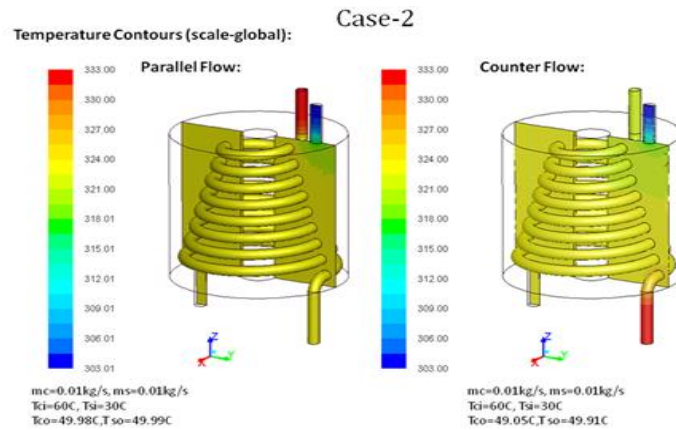
Nomenclature		Parallel Flow							
		Case-1		Case-2		Case-3		Case-4	
		Testing	CFD	Testing	CFD	Testing	CFD	Testing	CFD
Coil side fluid inlet temp	Tci (Hot Water)	29	29	60	60	65	65	51	51
Coil side fluid outlet temp	Tco (Hot Water)	28	24.5	43	49.98	61	56.42	45	39.35
Shell side fluid inlet temp	Tsi (Cold Water)	20	20	30	30	31	31	30	30
Shell side fluid outlet temp	Tso (Cold Water)	22	24.5	37	49.99	49	56.4	35	39.32
Coil side mass flow rate	mc	0.01		0.002		0.03562		0.0356	
Shell side mass flow rate	ms	0.01		0.001		0.012		0.0444	

Nomenclature		Counter Flow							
		Case-1		Case-2		Case-3		Case-4	
		Testing	CFD	Testing	CFD	Testing	CFD	Testing	CFD
Coil side fluid inlet temp	Tci (Hot Water)	29	29	60	60	65	65	51	51
Coil side fluid outlet temp	Tco (Hot Water)	28	23.73	43	49.05	61	54.99	45	36.43
Shell side fluid inlet temp	Tsi (Cold Water)	20	20	30	30	31	31	30	30
Shell side fluid outlet temp	Tso (Cold Water)	22	25.2	37	49.91	49	60.12	35	40.74
Coil side mass flow rate	mc	0.01		0.002		0.03562		0.0356	
Shell side mass flow rate	ms	0.01		0.001		0.012		0.0444	

Temperature contours for all the cases for parallel and counter flow:





## 5.0 CONCLUSION

A good agreement is found for temperatures obtained by simulation and measured by experimentation. Temperature variation of fluid through coil and shell is observed from contour the counter flow heat exchanger heat transfer is more. As the computational analysis is having more advantages such as cheaper, parametric study, less time than the experimental analysis. Present study provides the methodology for the helical coil heat exchanger by computationally which can be used for the analysis of helical coil heat exchanger using computational method. Heat transfer coefficients correlation for the helical coil can be developed which gives satisfactory result over large range.

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